

A novel design in output tracing for nonlinear systems via the first and second time sliding mode control

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Abstract: The problem of output tracing for nonlinear, non-minimum phase systems will be surveyed in this article. A cascade form control structure will be presented, for the desirable output tracing during the stabilization of inner dynamics in a limited time that conjugates the first and second times of sliding mode methods together. This method is converged under the complete asymptotically feedback. Comparing to the other methods, this method main efficiency is that sliding mode dynamics of output tracing error variable has a lower rank and its adjustment is simpler consequently therefore the transient response variables have better properties. Theoric analysis and simulation results reveal the suggested method effectivness.

Keywords: output tracing, nonlinear non-minimum systems, sliding mode.

I. Introduction

In this study the lateral output detection problem will be studied for a category of undefined nonlinear systems that the output reference defined by an undefined nonlinear outer system with polynomial defined indicator. The suggested method improves the causality but it has the theoretical nature. Many efforts have been applied to address many of the issues mentioned above. (Cavallo and Natale, 2014) Several control techniques have been proposed for the noncausal case where the tracking reference profile is assumed to be known beforehand. An approximate solution for a special class of systems and trajectories is proposed in. Exact tracking of a known trajectory given by a noncausal system is achieved via a stable nonlinear inverse in. (Laghrouche et al, 2014) In the authors address the problem of asymptotic output tracking for a class of nonlinear uncertain systems, where the output reference profiles are defined by an unknown linear exosystem with known characteristic polynomial. The proposed method improves the causality with respect to the existing state of art, but the assumption that the characteristic polynomial of the exogenous system is known makes its impact of mainly theoretical nature. An extension to the result of has been proposed in, where the exogenous system, responsible for generating the output reference profile, is assumed to be unknown, but of given order, and its characteristic polynomial is identified on-line via a higher-order sliding mode (HOSM) parameter observer and it is used for generating the reference profile for the internal state. A restriction of the method proposed in is the assumption of internal state availability that was later overcome in by designing a suitable observer. In, internal state observation was tackled by including the presence of unknown inputs. (zare and kofigar, 2015)

The lots of attempts have been done, solving the aforementioned problems. Some control methods were suggested for the nonobvious cases, assumed that the detection reference is determined in advance. (Gao and chen, 2007) An approximated solution is suggested for an especial class of systems and movement trajectories.

In this study a novel double loop cascade like control sketch is presented, combining the first and second order SMC methods. The ESSC method will be used for the calculation of inner unstable dynamics limited solution. The suggested solution protects the convergence and stability of current methods while it also makes the limited convergence time possible for the inner dynamic situations, caused high optimization. (Karamimolaei et al, 2009) Moreover an undefined high frequency control matrix is considered in present study whereas in previous studies it was assumed that the matrix is completely defined.

Problem Formulation

$$y = G_p(s)[u + d_e(y, t)], \quad \square \square \square$$

where u is the control input, y is the output, $d_e(y, t)$ is a matched input disturbance and $G_p(s) = k_p(N_p(s)/D_p(s))$, with $N_p(s)$ and $D_p(s)$ being monic polynomials of degree m and n , respectively. The following assumptions are made:

- (A1) $G_p(s)$ is minimum phase, strictly proper and its parameters are unknown but belong to a known compact set.
- (A2) The degree n of $D_p(s)$ is a known constant.
- (A3) $G_p(s)$ has

known relative degree $n^* := n - m$. The above Assumptions(A1)–(A3) are usual in adaptive control [15]. Consider the following additional assumptions:

(A4) The sign of the high frequency gain $k_p \neq 0$ is unknown. (A5) The disturbance $d_e(y, t)$ is locally Lipschitz in $y, \forall y$, and piecewise continuous in $t, \forall t$. (A6) The nonlinear disturbance $d_e(y, t)$ satisfies $|d_e(y, t)| \leq \bar{d}_e(y, t), \forall (y, t)$

imposed on d_e , e.g., $d_e(y, t) = y^2$. Since finite-time escape is not precluded, a priori, $[0, t_M)$ is defined as the maximum time interval of definition of a given solution, where t_M may be finite or infinite.

Reference Model: the reference model is given by

$$y_m = M(s)r = (k_m/D_m(s))r, k_m > 0, \quad (2)$$

where the reference signal $r(t)$ is assumed piecewise continuous and uniformly bounded, D_m is a monic polynomial of degree n^* .

Control Objective: the control objective is to achieve global or semi-global stability and convergence of the error state with respect to the origin of the error space. In particular, the tracking error

$$e_0(t) = y(t) - y_m(t) \quad (3)$$

should asymptotically tend to zero, i.e., exact tracking is required. (Cavallo and Natale, 2014)

II. Inner loop designing

In the inner loop designing it is assumed that an undefined outer loop gives a vector signal as $v(t) \in \mathbb{R}^{(n-r)}$, that its time derivative has an upper bound depends on state and time.

Considering the usual model reference adaptive control (MRAC) approach, the output error e_0 satisfy (Hsu et al, 1994)

$$e_0 = k^* M(s)[u - u^*], \quad (4)$$

where $k^* = k_p/k_m$,

$$u^* := \theta^{*T} \omega - W_d(s) * d_e, \quad (5)$$

The signal u^* will be regarded as a matched input disturbance, thus an upper bound will be required. Since W_d is a proper stable transfer function and d_e satisfies Assumption (A6), then applying (Costa and Cunha, 2003) to the convolution $W_d(s) * d_e(y, t)$, one can find positive constants c_d, γ_d such that $|W_d(s) * d_e(y, t)| \leq \hat{d}_e(t)$, where \hat{d}_e is defined by

$$d_e(t) := \bar{d}_e(y, t) + c_d e^{-\gamma_d t} * \bar{d}_e(y, t). \quad (6)$$

Thus, from (5), u^* satisfies

$$|u^*(t)| \leq \bar{\theta}^T |\omega(t)| + \hat{d}_e(t), t \in [0, t_M). \quad (7)$$

Consider the case of relative degree one, unknown $\text{sgn}(k_p)$, and nonlinear disturbances. This section will generalize the results of (Yan et al, 2003) developed for linear plants.

The control law is defined by

$$\begin{aligned} u &= [u^+ = -f(t) \text{sgn}(e_0), t \in T^+, \\ u^- &= f(t) \text{sgn}(e_0), t \in T^-, \end{aligned} \quad (8)$$

where an appropriate monitoring function of the tracking error e_0 is used to decide when u would be switched from u^+ to u^- and vice versa, allowing the detection of any wrong estimate of $\text{sgn}(k_p)$. The sets T^+ and T^- satisfy $T^+ \cup T^- = [0, t_M)$ and $T^+ \cap T^- = \emptyset$, and as will be shown in the following analysis, both T^+ and T^- have

the form $[t_k, t_{k+1}) \cup \dots \cup [t_j, t_{j+1})$. Here, t_k or t_j denotes the switching time for u and will be defined later. We refer to such switchings as control sign switchings.

According to (4), the modulation function $f(t)$ should be a norm bound of u^* . From (7), one possible choice is

$$f(t) = \bar{\theta}^T |\omega_0| + \hat{d}_c(t) + \delta, \quad (9)$$

where δ is an arbitrary nonnegative constant. Consider for simplicity $M(s) = k_m/(s + a_m)$ ($a_m, k_m > 0$). Then for $\text{sgn}(k_p)$ known, one chooses the control u^+ or u^- , according to $k_p > 0$ or $k_p < 0$, respectively. Now, e_0 satisfies

$$\dot{e}_0(t) = -a_m e_0(t) + k_p [u(t) - u^*(t)] + \pi(t) \quad (10)$$

where $\pi(t)$ denotes a transient term due to initial conditions of the observable but not controllable subsystem of the nonminimal realization (A_c, b_c, h_c^T) of $M(s)$ in (4), used in MRAC theory [15]. Now, noting that $\text{sgn}(u - u^*) = -\text{sgn}(e_0)$, if the correct control direction is used and $f(t) > |u^*|$, then by using the *Comparison Theorem* [13], $|e_0|$ is bounded by the solution of the following differential equation

$$\dot{\xi}(t) = -a_m \xi(t) + \pi(t), \quad \forall t \in [t_0, t_M), \quad \xi(t_0^-) = e_0(t_0^-) \quad (11)$$

i.e., $\forall t \geq [t_0, t_M)$, one has

$$|e_0(t)| \leq |\xi(t)| \leq e^{-a_m(t-t_0)} |e_0(t_0^-)| + c_0 e^{-\delta t}, \quad (12)$$

where t_0 denotes some initial time.

Based on (12), consider the auxiliary function ϕ_k defined as follows:

$$\phi_k(t) = e^{-a_m(t-t_k)} |e_0(t_k)| + (k+1)e^{-tk+1}, \quad (13)$$

$t \in [t_k, t_M), t_0 := 0, (k = 0, 1, \dots)$.

The monitoring function ϕ_m can be defined as

$$\phi_m(t) := \phi_k(t), \quad \forall t \in [t_k, t_{k+1}) \in [0, t_M) \quad (14)$$

The motivation behind the introduction of ϕ_m is that π is not available for measurement. Reminding that the inequality (12) holds if the $\text{sgn}(k_p)$ is correctly estimated, it seems natural to use ξ as a benchmark to decide whether a switching of u is needed. However, since π is not available, one has to use ϕ_m to replace ξ and invoke the switching of ϕ_m . Note that from (14), one always has $|e_0(t_k)| < \phi_k(t_k)$ at $t = t_k$.

Hence, the switching time t_k for u from u^- to u^+ (or u^+ to u^-) is well-defined (for $k \geq 0$):

$$t_{k+1} = \begin{cases} \min\{t > t_k: |e_0(t)| = \phi_k(t)\}, & \text{if it exists} \\ t_M, & \text{otherwise} \end{cases} \quad (15)$$

2.1. Main Result for $n^* = 1$

Theorem 1: Assume that (A1)–(A6) hold. Consider the system defined by (1), (2) and (8) and the modulation function given in (9). Then, the control sign switchings, driven by the monitoring function (14), will stop after a finite number of switchings and both the tracking error e_0 and the complete state X_c will converge to zero at least exponentially.

Proof: We only sketch the proof, which is divided into three parts. First it is proved that the switching stops after a finite number of switchings (avoiding finite-time escape), since for some finite k^* the term $(k^*+1)e^{-t/(k^*+1)}$ of (13) will allow $\phi_k(t)$ to be an upper bound valid for ξ , in (12), consequently no switching will occur after that. Second if the control direction is correctly estimated or not, since ϕ_k converges to zero exponentially $e_0(t)$ will also converge to zero, at least exponentially, avoiding finite-time escape. Finally, the convergence of the complete error state X_c can be shown by using the regular form for the state space realization of (4).

Corollary 1: In Theorem 1, the control sign switching stops at a correct sign corresponding to the unknown sign of the control direction of the plant, i.e., for $t > t_{k^*}$, $u = u^+$, if $k_p > 0$ and $u = u^-$, otherwise.

Proof: The proof is based on a reverse dynamics argument. We know that if the sign is correct all trajectories of the system converge to the origin of the error state space.

Reverse Dynamics Argument: Assume that the final control sign is incorrect. Then, if we reverse the time, i.e., $t \rightarrow -t$, the resulting equations have the same stability properties as those obtained with the right control sign and thus all trajectories from any initial condition would converge to the origin, i.e., the origin would be a global sink in reverse time. Thus, in forward time, all trajectories not at the origin would diverge unboundedly. This is a contradiction, since by Theorem 1 the state converges to the origin. Thus, the ultimate control sign must be correct

III. Outer loop designing

Consider the internal dynamics bounded to manifold $\sigma = 0$. The main idea for generalizing the previous case consists in reducing the problem to the $n^*=1$ case by the introduction of the operator

$$L(s) = s^N + a_{N-1}s^{N-1} + \dots + a_0, \quad N := n^* - 1, \quad (16)$$

such that $G_p L(s)$ be of relative degree one (or, equivalently, almost strictly positive real –ASPR) and $M L(s)$ be SPR (or ASPR). However, $L(s)$ is non-causal and what can be actually implemented is an approximate realization of this operator. One approximation is L given by the linear lead filter

$$L(s) = L(s)/F(s), \quad F(s) = (\tau s + 1)^N \text{ and } \tau > 0, \quad (17)$$

As will be shown, this approximation leads to global/semiglobal stability with respect a residual set of order $O(\tau)$. However, it is well known that such filters usually lead to control chattering and nonzero residual tracking error due to the phase lag introduced by the time constant (τ). Alternatively,

$L(s)$ can be implemented by using the Levant's robust exact differentiators (RED) (Levant, 2003) which potentially allows the exact estimate of the e_0 derivatives. The problem is that such differentiators are valid only locally and may lead to unstable behavior with larger initial conditions (Nunes, 2004).

In the proposed control strategy, see Figure 1, $L(s)$ is replaced by a hybrid lead filter, named Global Robust Exact Differentiator (GRED). In Fig. 1, α represents a switching law. It is then possible to obtain an exact compensation of the relative degree while assuring global or semi-global stability properties of the closed loop system. The control sign is adjusted according to the monitoring function ϕ_m , as indicated in Fig. 1.

The control u is defined as in (8), replacing e_0 by $\tilde{\varepsilon}_0 := \alpha \tilde{\varepsilon}_0 + (1 - \alpha)\varepsilon_0$ (see Fig. 1), i.e.,

$$u = [u^+ = -f(t) \operatorname{sgn}(\tilde{\varepsilon}_0), t \in T^+, \\ u^- = f(t) \operatorname{sgn}(\tilde{\varepsilon}_0), t \in T^-,] \quad (18)$$

The strategy for switching the control direction, according to a new monitoring function ϕ_m , will be redefined later on.

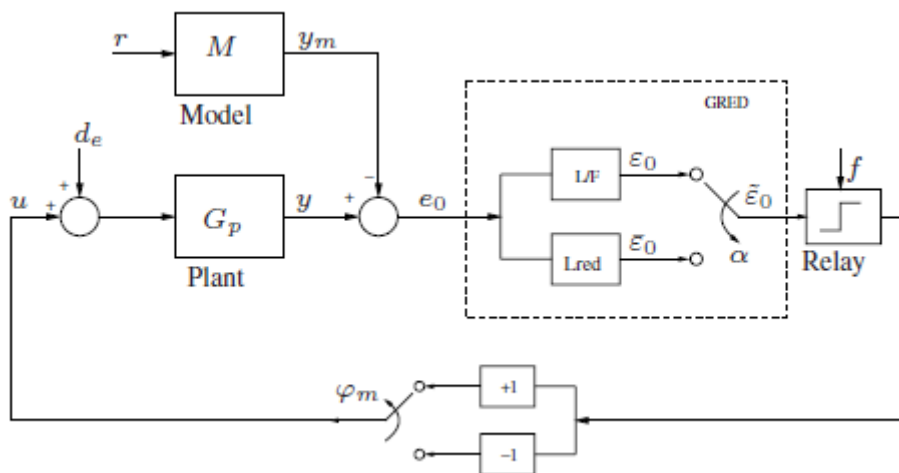


Fig1. Suggested Cascade-like controlling structure

IV. Auxiliary Errors for Analysis and Design

As explained above, assume that only the linear lead filter is active, i.e., $\tilde{\varepsilon}_0 = \varepsilon_0$. Then, from Figure 1, one has

$$\varepsilon_0 = \frac{L(s)}{F(\tau s)} e_0 \quad (19)$$

which can be rewritten as

$$\varepsilon_0 = k^* M L [u - u^*] + \beta_U + e^0_F, \forall t \in [0, t_M] \quad (20)$$

where

$$\beta_U := k^* M L_{(s)} [1 - F(\tau s)] F^{-1}(\tau s) * (u - u^*) \text{ and } (21)$$

$$|e_0^F| \leq R_1 e^{-\lambda_c t} + R_2 / \tau^{Ne-t/\tau} \leq R_a e^{-\lambda_a(t-t_e(\tau))}. \quad (22)$$

The positive constants R_1, R_2, R_a and Type equation here. λ_c are independent of $\tau > 0$; λ_c is lower than the stability margin of A_c and $0 < \lambda_a < \min(\lambda_c, 1/\tau)$, with $\tau > \tau$.

The first inequality in (22) holds $\forall t \geq 0$, while the last one holds only $\forall t \geq t_e$ where t_e is the peak extinction time, i.e., the smallest time value at which the inequality

$$R_2 / \tau^{Ne-t/\tau} \leq R_2, \forall t \geq t_e(\tau), \forall R_2 \text{ is satisfied for a fixed value of the parameter } \tau \in (0, 1).$$

The constants R_1 and R_2 are linear combination of the initial conditions $X_e(0)$ and $x_f(0)$, where x_f is the state vector of the realization $(A_f/\tau, B_f/\tau, C_f/\tau^N, 1/\tau^N)$ with $(A_f, B_f, C_f, 1)$ being the canonical controllable realization of L/F in (19).

By using this realization, peaking appears only in the output ε_0 while the state x_f is peaking free.

4.1. An Upper Bound for t_e (peak extinction time):

It can be easily concluded that $t_e(\tau)$ is uniformly bounded by a class-K function of τ . Moreover, there exist $\bar{t}_e(\tau) \in K$ such that

$$t_e(\tau) \leq \bar{t}_e(\tau), \quad (23)$$

which can be obtained from the known upper bounds of the plant parameters. Considering the error system (4), (19), the following state vector z is used

$$z^T := [X_e^T, x_f^T], z \in \mathbb{R}^{3n-2+N} \quad (24)$$

The following inequality is a consequence of the continuity of the Filippov solutions and the particular state realization associated with x_f :

$$|z(t)| \leq k_{z0} |z(0)| + V(\tau), \quad \forall t \in [0, t_e(\tau)] \subset [0, t_M], \forall \tau \in (0, \tau_1]; 0 < \tau_1 \leq 1; V \in K \text{ and } k_{z0} > 0 \text{ is a constant.} \quad (25)$$

4.2. Monitoring Function ($n^* > 1$)

The following lemma provides an upper bound for $|\varepsilon_0|$, valid if $\text{sgn}(k_p)$ is known and $t \in [t_e, t_M)$, from which the new monitoring function will be defined.

Lemma 1: Consider the I/O relationship

$$\varepsilon(t) = \bar{M}(s)[u + d(t)] + \pi(t) + \beta(t), \quad (26)$$

and any arbitrary initial time $\bar{t}_0 \geq 0$, where $\bar{M}(s) = \bar{k}/(s + \bar{\alpha})$ ($\bar{k}, \bar{\alpha} > 0$), $d(t)$ is LI, $\beta(t)$ and $\pi(t)$ are absolutely continuous, $\forall t \in [\bar{t}_0, t_M)$. Assume that $|\pi(t)| \leq R e^{-\lambda(t-\bar{t}_0)}$, $\forall t \in [\bar{t}_0, t_M)$, where R, λ are positive constants. If $u = -f(t) \text{sgn}(\varepsilon)$, where the modulation function $f(t)$ is LI and satisfies $f(t) \geq |d(t)|$, $\forall t \in [\bar{t}_0, t_M)$, then the signal $\bar{e}_{(t)} := \varepsilon(t) - \beta(t) - \pi(t)$ is bounded by (for any arbitrary t_i such that $\bar{t}_0 \leq t_i < t_M$ and $\bar{\alpha}\lambda := \min(\bar{\alpha}, \lambda)$)

$$|\bar{e}(t)| \leq |\varepsilon(t_i) - \beta(t_i) - \pi(t_i)| e^{-\bar{\alpha}(t-t_i)} + R e^{-\bar{\alpha}\lambda(t-\bar{t}_0)} + \prod \beta_i, t_0 \bar{\Pi}. \quad (27)$$

Reminding that $\varepsilon_0 = \beta_U + e_0 + e_F^0$ then $|\varepsilon_0| \leq |\beta_U| + |e_0| + |e_F^0|$. Now, applying Lemma 1 to (20), considering $t_0 := t_e$ and $ML(s) = k_m/(s + a_m)$ (for simplicity), and from (22) one has $\forall t, t_k$ such that $(t_M > t \geq t_k \geq t_e)$,

$$|\varepsilon_0(t)| \leq (|\varepsilon_0(t_k)| + |\beta_U(t_k)|)e^{-am(t-t_k)} + (2R_a e^{\lambda_a t_e})e^{-\lambda_a t} + 2\int_{t_e}^t (\beta_U(\tau))e^{-\lambda_a(t-\tau)} d\tau, \quad (28)$$

where $\lambda_a = \min\{a_m, \lambda_a\}$. Note that, according to Lemma 1, (28) is valid for the modulation function $f(t)$ given in (9). Consider the available signal

$$\beta_U = 2k \cdot \tau W_\beta(s) \cdot f(t) \quad (29)$$

where $\tau W_\beta(s)$ is a first order approximation filter (FOAF, [19]) for the transfer function $ML(s) [1 - F(\tau s)] F^{-1}(\tau s)$. Note that, from (21), (18) and (9), one has $\beta_U(t) \leq \beta_U(t)$ ($\forall t \in [0, t_M]$). Let

$$\phi_k(t) := (|\varepsilon_0(t_k)| + \beta_U(t_k))e^{-am(t-t_k)} + a(k)e^{-\lambda_c t} + 2\int_{t_e}^t (\beta_U(\tau))e^{-\lambda_a(t-\tau)} d\tau, \quad (30)$$

$\forall t \in [t_k, t_M]$, with λ_c in (22) and $a(k)$ is any positive monotonically increasing unbounded sequence. The monitoring function for $n^* > 1$ ϕ_m is defined by

$$\phi_m(t) := \phi_k(t), \quad \forall t \in [t_k, t_{k+1}) \quad C[0, t_M]. \quad (31)$$

Note that ϕ_m is discontinuous in t . The switching time t_k for u from u^- to u^+ (or u^+ to u^-) is well-defined by: $t_{k+1} := \lceil \min\{t > t_k : |\varepsilon_0(t)| = \phi_k(t)\} \rceil$, if it exists, t_M , otherwise,

where $k \geq 1, t_0 := 0$ and $t_1 := t_e$. For convenience, $\phi_0 := 0, \forall t \in [t_0, t_1)$. The following proposition follows directly from the definition of the monitoring function ϕ_m , in (31).

Proposition 1: Let $k \geq 1$ be the largest switching index of the monitoring function (31), such that $t_k \in [0, t_M)$, then the auxiliary error $\varepsilon_0(t)$ is bounded by

$$|\varepsilon_0(t)| \leq \phi_m(t), \quad \forall t \in [t_1, t_M]. \quad (33)$$

V. Dynamic stability of sliding mode

Now we are going to analyze the stability of system path properties that are bounded to $\sigma = 0$ manifold and under the extra situation and using that, determine a suitable criterion for the selection of design matrix D .

$$(G_1 - G_2 D)e_{e1} + \varepsilon_c = G_2 H_v \quad (34)$$

The below equivalent dynamic will be obtained:

$$\begin{aligned} e \varepsilon_1 &= E_1 e \varepsilon_1 + E_2 e \varepsilon_2 \\ &= (E_1 - E_2 (D + H(G_2 H)^{-1} (G_1 - G_2 D))) e \varepsilon_1 \\ &\quad - E_2 H(G_2 H)^{-1} \varepsilon_c = M e \varepsilon_1 - E_2 H(G_2 H)^{-1} \varepsilon_c \end{aligned} \quad (35)$$

It describes the system movement in sliding mode situations that $\sigma = 0$ and $\dot{e}_\eta = e_\eta = 0$ simultaneously. The last term of input is bounded constantly that converges to zero asymptotically which doesn't affect the asymptotic stability. This depends on D and H matrices for the below matrix from 22.

$$M = E_1 - E_2 (D + H(G_2 H)^{-1} (G_1 - G_2 D)) \quad (36)$$

Rewrite the relation 23 as below:

$$M = E_1 - E_2 H(G_2 H)^{-1} G_1 + E_2 (H(G_2 H)^{-1} G_2 - I)^D \quad (37)$$

Let's consider another assumption.

Assumption 5: can find matrix H so the below matrices create a controllable pair.

$$E_1 - E_2 H(G_2 H)^{-1} G_1 \quad \text{and} \quad E_2 (H(G_2 H)^{-1} G_2 - I)$$

Considering the upper assumption, it could be resulted that designed matrix D can be chosen randomly for the putting of especial amount of matrix M in relation 24. It must be mentioned that assumption 5 is necessary but isn't enough. But in this way the M range couldn't be assigned.

VI. Simulation Results

The suggested algorithm usefulness is revealed by the simulation. A non-minimum 5 order system MIMO phase is considered, stimulated by two harmonic signal and the system is solved using the below.

This section presents an illustrative simulation example which highlights the performance of the proposed control scheme for a nonlinear plant with relative degree $n^* = 3$.

Example 1: Consider an open-loop unstable plant with transfer function given by:

$$G_p(s) = \frac{1}{(s+2)(s+1)(s-1)}$$

Being controlled by the VS-MRAC controller of Figure 1 and under the action of a nonlinear input disturbance $d_e(y, t) = y^2 + \text{sqw}(5t)$, where sqw denotes a unit square wave. The reference model is $M(s) = \frac{4}{(s+2)^3}$ and the linear

lead filter is given in (17) with $L(s) = (s+2)^2$ and $\tau = 0.01$. The monitoring function is obtained from (31) with $a(k) = k+1$ and $\lambda c = 0.5$. The plant initial conditions are $y(0) = 2, y'(0) = 0$ and $y''(0) = 2$ and the feedback is positive at $t = 0$ (wrong control direction).

Figure 2 corresponds to a simulation result when the reference signal is a sinusoid of amplitude 1 and frequency 1 rad/s. The convergence of the plant output signal to the model reference output is clear. Figure 3 (a) shows that just one switching in the control sign was needed (first jump of ϕ_m when it meets $\tilde{\epsilon}_0$). After that, the control direction is correctly identified and the auxiliary error $\tilde{\epsilon}_0$, as well as the tracking error, vanish in finite time. Note that the second discontinuous-like change of ϕ_m is not due to a change between u^+ and u^- . It is due to the $\prod(\beta_U)_i$ term in (30), led to quite reasonable transient behavior in our simulations in contrast to the Nussbaum gain approach.

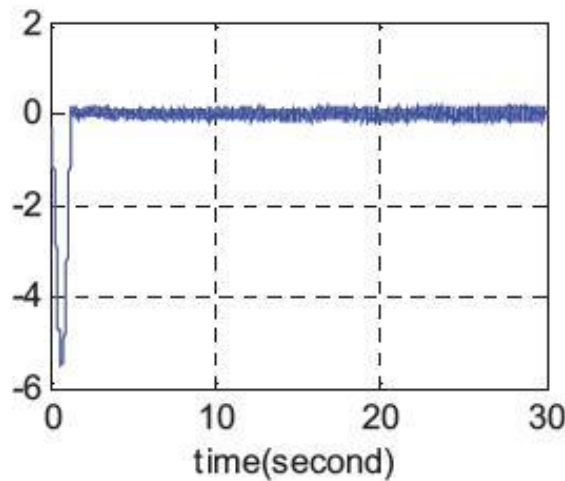


Fig 2- First output vector ξ_1 (in complete scale)

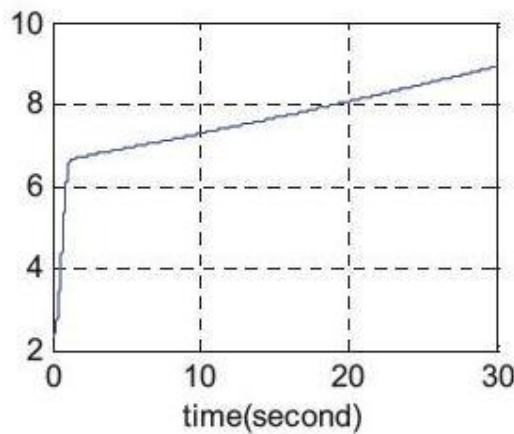


Fig 3- First output vector ξ_1 (maximized)

The figures 2 to 4 show the performance, high accuracy and authenticity of controller output detection. Output profile discontinuity affects the both ESSC filter and internal dynamic detection (see figure 5). Finally the figures 6 show the assessment of sliding variable boundary layer for outer loop σ .

VII. Conclusion

The outer control detection was solved for a set of nonlinear non-minimum phase systems, using the first/second order hybrid method of sliding mode. Despite of using the detector in present situations, the creation of one solution that doesn't need the knowing of Q matrix is under study. This is an important and controversial problem that needs the complete review in ESSC method.

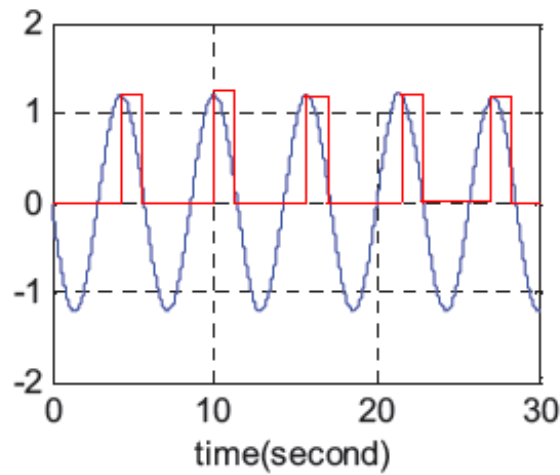


Fig4. Second output vector ξ_2 (maximized)

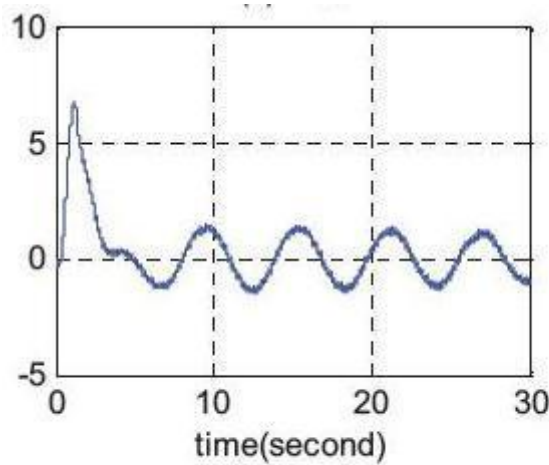


Fig5. Internal dynamic η and SSC filter performance, means the θ input and $\hat{\eta}_c$ output.

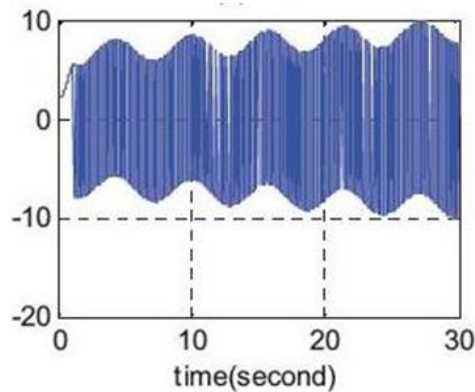


Fig6. Sliding variable σ of internal control loop

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